

Registration No : **Bs Tech(M)/3-19/M011**

Student Name : **AHMED RAZA**

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Title of Course : **Heat Transfer**

Name of Faculty member : **Sir WAQAR HAYDER**

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Q3(A)

ANSWER:

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THERMAL CONTACT CONDUCTANCE”:

In physics, thermal contact conductance is the study of heat conduction between solid bodies in thermal contact. The thermal contact conductance coefficient, h_c , is a property indicating the between two bodies in contact. The inverse of this property is termed thermal contact resistance.

According to Fourier's Law, the heat flow between the bodies is found by using the relation:

$$Q = -KA \frac{dT}{dx}$$

where q is the heat flow, k is the thermal conductivity, A is the cross-sectional area and dT/dx is the temperature gradient in the direction of flow.

From considerations of energy conservation, the heat flow between the two bodies A and B, is found:

One may observe that the heat flow is directly related to the thermal conductivities of the bodies in contact, k_A and k_B , the contact area A , and the thermal contact resistance, $1/h_c$ which as previously noted, is the inverse of the thermal conductance coefficient, h_c .

Most experimentally determined values of thermal contact resistance fall between 0.000005 and 0.0005 m²/W (the corresponding range of thermal contact conductance is 200000 to 2000 W/m² K). To know whether the thermal contact resistance is significant or not, magnitudes of the thermal resistance of the layers are compared with typical values of thermal contact resistance. Thermal contact resistance is significant and may dominate for good heat conductors such

as metals but can be neglected for poor heat conductors such as insulators. some of the fields where contact conductances of importance are:

Q3(b)

Solua: Q3 (b)

The overall heat flow is subject to three thermal resistance one conduction resistance for each bar and the contact resistance for the bars.

$$R_{tn} = \frac{\Delta x}{KA} = \frac{(0.1)(4)}{(16.3)\pi(3 \cdot 10^{-2})^2} = 8.679^\circ\text{C}$$

from table 2-2 the contact resistance

$$R_c = \frac{1}{hcA} = \frac{(5.28 \times 10^{-4})(4)}{\pi(3 \times 10^{-2})^2} = 0.747 \text{ } \uparrow \text{ } \text{f e w}$$

The total thermal resistance is therefore

$$\sum R_{tn} = (2)(8.679) + 0.747 = 18.105$$

and the overall heat flow is -

$$q = \frac{\Delta T}{\sum R_{tn}} = \frac{100}{18.105} = 5.52 \text{ W} \quad [18.83 \text{ Btu}]$$

The temperature drop across the contact is found by taking the ratio of the contact resistance to the total thermal resistance.

Q2(b)

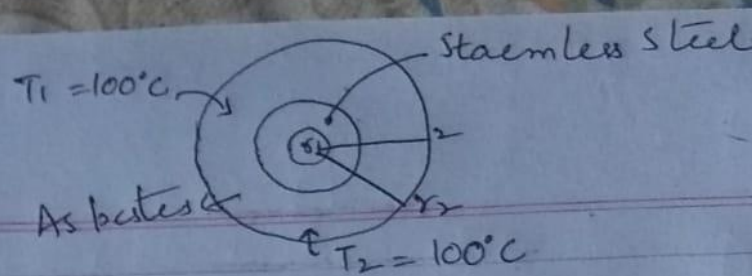


Diagram showing a cylindrical pipe with an inner radius r_1 , an insulation layer with outer radius r_2 , and an outer stainless steel layer with outer radius r_3 . The inner fluid temperature is $T_1 = 100^\circ\text{C}$ and the outer fluid temperature is $T_2 = 100^\circ\text{C}$. The insulation is labeled "Asbestos" and the outer layer is labeled "Stainless steel".

$$\frac{T_1}{\frac{\ln(r_2/r_1)}{2\pi K_1 L}} = \frac{T_2}{\frac{\ln(r_3/r_2)}{2\pi K_2 L}}$$

$$\frac{2}{L} = \frac{2\pi(T_1 - T_2)}{\ln(r_2/r_1)K_1 + \ln(r_3/r_2)K_2}$$

$$= \frac{2\pi(600 - 100)}{(\ln^2)119 + (\ln^2/2)/0.2}$$

$$= 680 \text{ w/m}$$

This heat flow may be used to calculate the interface temperature between the outside tube wall and the insulation we have

$$\frac{q}{L} = \frac{T_a - T_2}{\ln(r_3/r_2)/2\pi K_2} = 680 \text{ w/m}$$

When T_a is the interface temperature which may be obtained as

$$T_a = 595.8^\circ\text{C}$$

The largest thermal resistance clearly result from the insulation and the major portion of the temperature drop is through that inertial.

Q1(b)

Q1(b)

The total heat loss is the sum of convection and radiation from table 1-3 we see that an estimate for the heat-transfer coefficient for free convection with this geometry and air is $h = 6.5 \text{ W/m}^2 \cdot ^\circ\text{C}$. The surface area is πdL , so the convection loss per unit length is

$$q/L)_{\text{conv}} = h (\pi d) (T_w - T_\infty) \\ = (6.5) (\pi) (0.05) (50 - 20) \\ = 30.63 \text{ W/m}$$

The pipe is a body ~~surrounding~~ surrounded by a large enclosure so the radiation heat transfer can be calculated from equation (1-12) with $T_1 = 50^\circ\text{C} = \text{K}$ and $T_2 = 20^\circ\text{C} = 293\text{K}$, we have

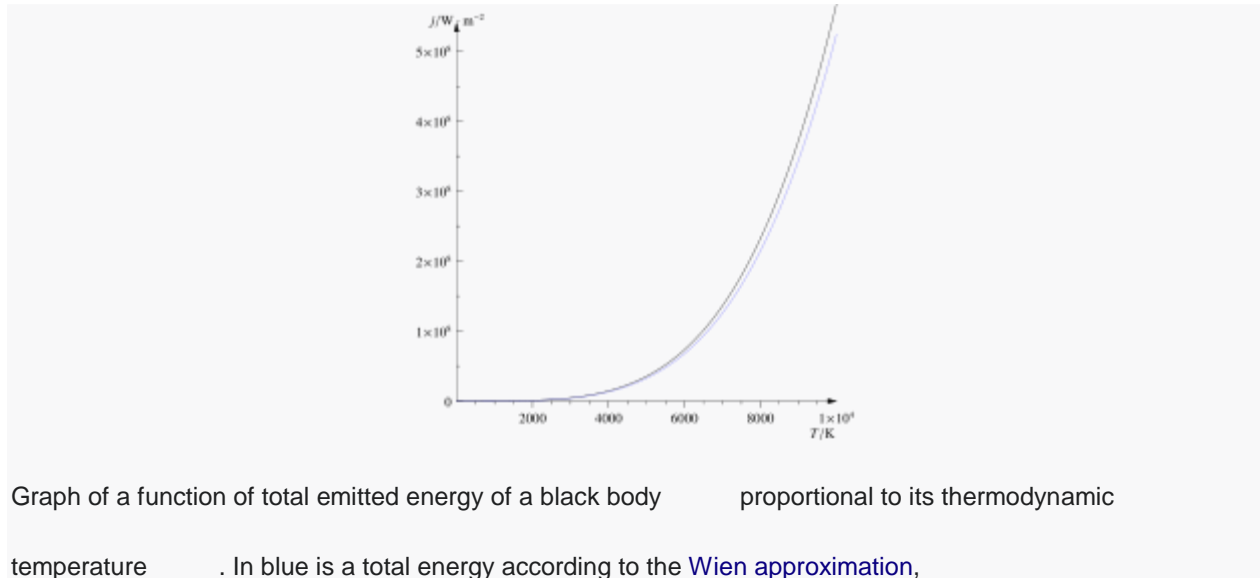
$$\Delta T_c = \frac{R_c}{\sum R_{tn}} \Delta T = \frac{(0.47)}{18.105} (100) = 4.13^\circ\text{C}$$

$$[39.43^\circ\text{F}]$$

In this problem the contact resistance represent about 4 percent of the total resistance.

Q1(A)

See also: *Black body, Black-body radiation, Planck's law, and Thermal radiation*



The **Stefan–Boltzmann law** describes the power radiated from a **black body** in terms of its **temperature**. Specifically, the Stefan–Boltzmann law states that the total **energy** radiated per unit **surface area** of a **black body** across **all wavelengths** per unit **time** (also known as the black-body *radiant emittance*) is directly **proportional** to the fourth power of the black body's **thermodynamic temperature** T :

The **constant of proportionality** σ , called the **Stefan–Boltzmann constant**, is derived from other known **physical constants**. The value of the constant is

where k is the **Boltzmann constant**, h is **Planck's constant**, and c is **the speed of light in a vacuum**. The **radiance** (watts per square metre per **steradian**) is given by

A body that does not absorb all incident radiation (sometimes known as a grey body) emits less total energy than a black body and is characterized by an **emissivity**, ϵ :

The radiant emittance has **dimensions** of **energy flux** (energy per unit time per unit area), and the **SI units** of measure are **joules** per second per square metre, or equivalently, **watts** per square metre. The SI unit for absolute temperature T is

the **kelvin**. ϵ is the **emissivity** of the grey body; if it is a perfect blackbody, $\epsilon = 1$. In the still more general (and realistic) case, the emissivity depends on the wavelength, λ .

To find the total **power** radiated from an object, multiply by its surface area, A :